## IN-DEPTH SURVEY REPORT:

# A LABORATORY EVALUATION OF PROTOTYPE ENGINEERING CONTROLS DESIGNED TO REDUCE OCCUPATIONAL EXPOSURES DURING ASPHALT PAVING OPERATIONS

at

Caterpillar Paving Products (Barber-Greene)

DeKalb, Illinois

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Public Health Service
Centers for Disease Control and Prevention
National Institute for Occupational Safety and Health
Division of Physical Sciences and Engineering
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PLANT SURVEYED

Caterpillar Paving Products

(Historical Name Barber-Greene)

12101 Barber-Greene Road DeKalb, Illnois 60115

SIC CODE

1611

SURVEY DATE

March 12-15, 1996

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#### **EXECUTIVE SUMMARY**

On March 12-15, 1996, researchers from the National Institute for Occupational Safety and Health (NIOSH) evaluated a prototype engineering control system at Caterpillar Paving Products, DeKalb, Illinois The control system was designed for the control of asphalt emissions from the auger area during asphalt paving. The Caterpillar engineering controls evaluation was completed as part of a Department of Transportation (DOT) project to evaluate the effectiveness of engineering controls on asphalt paving equipment. NIOSH researchers are conducting the research through an inter-agency agreement with DOT's Federal Highway Administration. Additionally, the National Asphalt Paving Association is playing a critical role in coordinating the paving manufacturers' and paving contractors' voluntary participation in the study.

The study consists of two major phases. During the primary phase, NIOSH researchers visited each participating manufacturer and evaluated their engineering control designs under managed environmental conditions. The indoor evaluation used tracer gas analysis techniques to both quantify the control's exhaust flow rate and determine the capture efficiency. Results from the indoor evaluations provided equipment manufacturers with the necessary information to maximize engineering control performance prior to the second phase of the study, performance evaluation of the prototype engineering controls under "real-life" paving conditions. The scope of this report is limited to the Caterpillar phase one evaluation.

The Caterpillar phase one evaluation studied the performance of one engineering control design using two different fans. Both fans were tested indoors and the larger fan was also tested outdoors. The control system design incorporated a long hood mounted on the back of the tractor above the auger area, covering approximately 60 percent of the area between the tractor and the screed. A duct mounted at the top of the slat conveyor connected the hood to a fan mounted under the tractor deck. The fan's exhaust duct extended six feet above the tractor deck. The control system exhaust volume was 1,120 cubic feet per minute (cfin) with the 1-0 horsepower (hp) fan and 1,350 cfin for the 1-5 hp fan. The average indoor capture efficiency was approximately 72 percent with the 1-0 hp fan and 95 percent with the 1-5 hp fan. The outdoor evaluation, using the 1-5 hp fan, revealed an average capture efficiency of 68 percent. Compared to the indoor, the outdoor results showed a 27 percentage point decline in capture efficiency and increased variation in results as wind gusts hampered the control's ability to consistently capture the surrogate contaminant.

The evaluated Caterpillar engineering control system has the potential to significantly reduce worker exposure during asphalt paving processes. The potential reduction is increased when using the larger exhaust fan. Recommendations to Caterpillar design engineers include. (1) Modifying both the transition between the duct and the hood, and the transition between the duct and the fan to reduce static pressure losses and increase exhaust flow rate, (2) Increasing the duct area located above the slat conveyors will also reducing the static pressure losses and increasing the exhaust flow rate, and (3) Increasing the extent of enclosure coverage around the auger area to reduce cross-draft interference and increase capture efficiency.

Since the intent of the phase one evaluations was to provide equipment manufacturers with engineering performance and design feedback, various original and imaginative approaches were developed with the knowledge that these prototypes would undergo preliminary performance testing to identify which designs showed the most ment. Each manufacturer received design modification recommendations specific to their prototypes' performance during the phase one testing. Prior to finalization of this report, each manufacturer received the opportunity to identify what modifications and/or new design features were incorporated into the "final" prototype design prior to the phase two evaluations. No further design information was provided for this report.

#### INTRODUCTION

The National Institute for Occupational Safety and Health (NIOSH), a Federal agency located in the Centers for Disease Control and Prevention under the Department of Health and Human Services was established by the Occupational Safety and Health Act of 1970. This legislation mandated NIOSH to conduct research and educational programs separate from the standard setting and enforcement functions conducted by the Occupational Safety and Health Administration (OSHA) in the Department of Labor. An important area of NIOSH research deals with methods for controlling occupational exposure to potential chemical and physical hazards.

The Engineering Control Technology Branch (ECTB) of the Division of Physical Sciences and Engineering (DPSE), has the lead within NIOSH to study and develop engineering controls and assess their impact on reducing occupational illness. Since 1976, ECTB has conducted a large number of studies to evaluate engineering control technology based upon industry, process, or control technique. The objective of each of these studies has been to identify or design engineering control techniques and to evaluate their effectiveness in reducing potential health hazards in an industry or at specific processes. Information on effective control strategies is subsequently published and distributed throughout the affected industry and to the occupational safety and health community.

### BACKGROUND

On March 12-15, 1996, researchers from the National Institute for Occupational Safety and Health (NIOSH) conducted an evaluation of a prototype engineering control system at Caterpillar Paving Products, DeKalb, Illinois—The control system was designed for the control of asphalt emissions from the auger area during asphalt paving—The NIOSH researchers included Leroy Mickelsen, Chemical Engineer, Gary Earnest, Industrial Engineer, and Walt Haag, Industrial Engineer, all from the NIOSH Engineering Control Technology Branch (ECTB), Division of Physical Sciences and Engineering (DPSE)—The DPSE researchers were primarily assisted by Jim Placiennik, a Caterpillar Design Engineer

The Caterpillar engineering control system evaluation was completed as part of a Department of Transportation (DOT) project to evaluate the effectiveness of engineering controls on asphalt paving equipment. NIOSH/DPSE researchers are conducting the research through an interagency agreement with DOT's Federal Highway Administration (FHWA). Additionally, the National Asphalt Pavement Association (NAPA) has played a critical role in coordinating the paving manufacturers' voluntary participation in the study. The study consisted of two major phases During the primary phase, NIOSH researchers visited each participating manufacturer and evaluated their engineering control designs under managed environmental conditions. [General protocols for the indoor evaluations are located in Appendix A. Minor deviations from these protocols may sometimes occur depending upon available time, prototype design, equipment performance, and available facilities.] Results from the phase one evaluations are provided to the

equipment manufacturers along with design change recommendations to maximize engineering control performance prior to the phase two evaluations. The second phase evaluations, which began in mid-1996, include a performance evaluation of the prototype engineering controls under "real-life" conditions at an actual paving site. The results from the Caterpillar phase two evaluation will be published in a separate report.

#### **DESIGN REQUIREMENTS**

When designing a ventilation control, the designer must apportion the initial design criteria among three underlying considerations, the level of enclosure, the hood design, and the available control ventilation. When possible, an ideal approach is to maximize the level of enclosure in order to contain the contaminant emissions. With a total or near-total enclosure approach, hood design is less critical, and the required volume of control ventilation is reduced. Many times, worker access or other process requirements limit the amount of enclosure allowed. Under these constraints, the designer must compromise on the level of enclosure and expend increased attention to hood design and control ventilation.

In the absence of a totally enclosed system, the hood design plays a critical role in determining a ventilation control's capture efficiency. Given a specified exhaust flow rate, the hood shape and configuration affect the ventilation control's ability to capture the contaminant, pull it into the hood, and direct it toward the exhaust duct. A well-engineered hood design strives to achieve a uniform velocity profile across the open hood face. When good hood design is combined with proper enclosure techniques, cross-drafts and other airflow disturbances have less of an impact on the ventilation control's capture efficiency.

In addition to process enclosure and hood design, a third area of consideration when designing a ventilation control, is the amount of ventilation air (volumetric flow and/or velocity) required to capture the contaminant and remove it from the working area. For most work processes, the contaminant must be "captured" and directed into the contaminant removal system. For ventilation controls, this is achieved with a moving air stream. The velocity of the moving air stream is often referred to as the capture velocity. In order to maintain a protected environment, the designed capture velocity must be sufficient to overcome process-inherent contaminant velocities, convective currents, cross-drafts, or other potential sources of airflow interference. The minimum required exhaust flow rate (Q) is easily calculated by inputting the desired capture velocity and process geometry information into the design equations specific to the selected hood design. Combining Q with the calculated pressure losses within the exhaust system allows the designer to appropriately select the system's exhaust fan

For most ventilation controls, including the asphalt paving controls project, these three fundamentals, process enclosure, hood design, and capture velocity are interdependent. A design, which lacks process enclosure, can overcome this shortcoming with good hood design and increased air flow. Alternatively, lower capture velocities may be adequate if increased enclosure and proper hood design techniques are followed. Additional information on designing

ventilation controls can be found in the American Conference of Governmental Industrial Hygienists' (ACGIH) Industrial Ventilation Manual [ACGIH, 6500 Glenway Avenue, Building D-7, Cincinnati, Ohio 45211]

#### **EVALUATION PROCEDURE**

The Caterpillar engineering control phase one evaluation was conducted in a large bay area within a separate research building removed from the manufacturing plant. A large overhead door provided access for the paver to be partially driven into the bay area. The paver was positioned in the doorway so that the screed and rear half of the tractor were within the bay area (referred to as the testing area). The front half of the tractor, the paver engine and its exhaust, and the control system's exhaust were all outside of the building. The overhead door was lowered to rest on top of the tractor, and the remaining doorway openings around the tractor were sealed to isolate the front and rear halves of the paver. During each test run, the engine exhaust and control system exhaust were discharged to the outside of the building. This setup proved very effective at preventing the engine exhaust, engine cooling air, and the captured surrogate contaminants from reentering the testing area.

A theatrical smoke generator produced smoke as a surrogate contaminant. The smoke was released through a perforated distribution tube. The tube placement traversed the width of the auger area between the tractor and the screed and rested on the ground under the augers. Initially, the smoke was used to observe airflow patterns around the paver and to observe capture by the control systems. (The general smoke test protocol is in Appendix A.) This test also helped to identify failures in the integrity of the barrier separating the front and rear portions of the paver. After sealing leaks within this barrier, smoke was again released to identify airflow patterns within the test area and to visually observe the control system's performances.

The second method of evaluation was the tracer gas evaluation. This evaluation was designed to (1) Calculate the total volumetric exhaust flow of each hood, and (2) Evaluate each hood's effectiveness in controlling and capturing a surrogate contaminant under the "controlled" indoor scenario. Sulfur hexafluoride ( $SF_6$ ) was the selected tracer gas. At the concentrations generated for these evaluations,  $SF_6$  behaves as a non-toxic, surrogate contaminant which follows the air currents of the ambient air in which it is released. Since  $SF_6$  is not naturally found within ambient environments, it is an excellent tracer gas for studying ventilation system characteristics. The general protocol for the tracer gas evaluation is in Appendix A.

A photo-acoustic infra-red detector (Bruel & Kjaer Model 1302) was calibrated in the NIOSH laboratories prior to the evaluation. Known amounts of reagent grade  $SF_6$  were injected into 12-liter Milar sampling bags and diluted with nitrogen to predetermined concentrations. Five concentrations ranging from 2 to 100 parts per million (ppm)  $SF_6$ /mitrogen were generated. A curve was fit to the data and used to convert detector response to  $SF_6$  concentrations. Calibration data are in Appendix B

To quantify exhaust flow rate, the tracer gas discharge tubes were placed directly into the exhaust ducts of the engineering control system. A known flow rate of  $SF_6$  was released into the duct(s) and the analytical instrument measured the concentration of  $SF_6$  in the control system's exhaust Measurements were taken downstream of the exhaust fan to allow for thorough mixing of the exhaust air stream. The exhaust flow rate was calculated using the following equation

$$Q_{(exh)} = \frac{Q_{(SF_6)}}{C_{(SF_6)}^+} \times 10^6$$
 Equation 1

where

 $\mathbf{Q}_{(exh)}$  = flow rate of air exhausted through the ventilation system (lpm or cfm)

 $\mathbf{Q}_{(SF6)}$  = flow rate of  $SF_6$  (lpm or cfm) introduced into the system

 $C^*_{(SF6)}$  = concentration of SF<sub>6</sub> (parts per million) detected in exhaust. And the indicates 100 percent capture of the released SF<sub>6</sub>

[To convert from liters per minute (lpm) to cubic feet per minute (cfm), divide lpm by 28 3 ]

To quantify capture efficiency, we released the SF<sub>6</sub> through distribution plenums. Each discharge hose fed from the SF<sub>6</sub> regulator, through a mass flow controller, and into a T-shaped distribution plenum. Each plenum was approximately 4' wide and designed to release the SF<sub>6</sub> evenly throughout its width. During the capture efficiency test, we placed the discharge plenums within the auger area between the paving tractor and the screed. A known quantity of SF<sub>6</sub> slowly discharged through the plenums into the auger area. A direct-reading analytical instrument measured the concentration of the tracer gas in the exhaust on the discharge side of the control. The capture efficiency was calculated using the following equation.

$$\eta = 100 \times \frac{\frac{C_{(SF_6)} \times Q_{(exh)}}{10^6}}{Q_{(SF_6)}}$$
 Equation 2A

where

 $\eta = capture efficiency$ 

 $C_{(SF6)}$  = concentration of  $SF_6$  (parts per million) detected in exhaust

 $\mathbf{Q}_{(exh)}$  = flow rate of air exhausted through the ventilation system (lpm or cfm)

 $\mathbf{Q}_{(SF6)}$  = flow rate of  $SF_6$  (lpm or cfm) introduced into the system

[To convert from liters per minute (lpm) to cubic feet per minute (cfm), divide lpm by 28 3 ]

**NOTE** When the flow rate of  $SF_6[Q_{(SF6)}]$  used to determine the engineering control's capture efficiency is the same as that used to quantify the exhaust flow rate, equation 2A may be simplified to

$$\eta = \frac{C_{(SF_6)}}{C_{(SP_c)}^*} \times 100$$
 Equation 2B

where the definitions for  $C^*_{(SF6)}$ ,  $\eta$ , and  $C_{(SF6)}$  remain the same as in equations 1 and 2A

Both flow rate and capture efficiency tests were repeated. The paver was shut down and background SF<sub>6</sub> measurements taken between trials. The exhaust flow rate of the control system was evaluated at two different paver idle speeds to determine its effect.

Since the Caterpillar engineering control design was tested using two different exhaust fans, the most effective system-fan combination, as determined by the indoor evaluation, was selected for further evaluation outdoors with the paver positioned in prescribed stationary orientations. The paver was randomly oriented in four different directions relative to the prevailing wind. Wind velocity measurements were taken, as well as exhaust flow rates and capture efficiency, during the outdoor evaluations. The outdoor stationary evaluation provided feedback on the sufficiency of the engineering control's hood enclosure for performance in an outdoor environment.

#### EQUIPMENT

(See Appendix A)

## ENGINEERING CONTROL DESIGN DESCRIPTION

The Caterpillar asphalt paving engineering control was a local exhaust ventilation system with no additional enclosures around the auger area. It consisted of a hood, duct, fan, and exhaust stack. The local exhaust ventilation system was designed and installed by engineers at Caterpillar. The control system was retrofitted to a Caterpillar Paver Model AP-1050 with an Extend-o-mat screed no 10-20B. The hood was located on the rear of the tractor, centered over the auger's drive train, and above the auger. The hood was approximately 65' wide. It extended approximately 13" past the rear of the tractor and then curved downward for approximately 6". The hood's size and position created a partial enclosure over the area where hot mix asphalt is delivered to the screw augers. Caterpillar engineers noted that during the asphalt paving process, workers prefer an unobstructed view into the auger area.

The hood was connected to a duct which ran horizontally from the auger to the fan. The cross sectional area of the entire duct was 72.5 square inches (1.25" by 58"). It was located directly above the slat conveyors. The slat conveyors are used to transport asphalt from the hopper (on the front of the paver) to the augers (on the rear of the paver). The duct was connected to the fan inlet. The fan was a high volume, direct drive, centrifugal blower that was manufactured by the Dayton Electric Manufacturing Company. The fan was located under the tractor deck next to the engine. Two different fans were used in this system during the survey. Initially, a 1.0 horsepower (hp) fan that operated at approximately 1,725 revolutions per minute (rpm) was used. During the second day of the study, a 1.5 hp fan, operating at the same rpm, was installed and evaluated.

The hydraulic fan motor was connected to a regulating valve feeding off of the tractor's hydraulic system. This valve enabled the fan to run at a relatively constant fan speed, independent of the engine idle speed. The fan exhausted to the atmosphere through an 8" diameter duct located just behind the main engine exhaust stack. The fan exhaust stack extended approximately 6' above the paver deck.

#### **DATA RESULTS**

#### Smoke Evaluations

The smoke test evaluation provided only qualitative information. After verifying the integrity of the separating barrier, smoke was re-released to identify airflow patterns within the test area and to visually observe the control system's performance. This information assisted the researchers in preparing the test area for the quantitative tracer gas evaluation.

## **Tracer Gas Evaluation**

(A copy of the tracer gas evaluation data files and associated calculations are included in Appendix B)

The calibration data from the B&K was used to convert the instrument's response to the actual  $SF_6$  concentration in sampled air. The following equation was derived from calibration data ranging from 0 to 60 ppm in Appendix B

$$SF_6$$
 Concentration = 403 -  $\sqrt{162,403}$  - 844\*Response

Where Response = the B&K detector response (ppm)

Evaluations conducted indoors are considered controlled conditions. Building pressure fluctuations and air currents from moving people or equipment are considered insignificant compared to outdoor conditions. The results are reported in Tables I and II in terms of an average and a range of the 6 to 10 measurements for each run. Multiple tests were performed for each fan resulting in an average exhaust flow rate of 1,120 cfm for the 1-0 hp fan and 1,350 cfm for the 1-5 hp fan. The average indoor capture efficiency was 72 percent with the 1-0 hp fan and 95 percent with the 1-5 hp fan. For comparison purposes, a pitot tube traverse of the ventilation system's exhaust duct resulted in a calculated average flow rate of 1,280 cfm for the 1-0 hp fan and 1,400 cfm for the 1-5 hp fan. The air velocity at the face of the hood ranged from 110 to 150 fpm

The outdoor evaluation occurred in a parking area. There were some large trucks in an adjacent lot which may have partially obstructed the wind. Wind gusted from 5 to 10 miles per hour (mph) with most readings averaging approximately 6 mph. Wind velocities were measured with a hot-wire anemometer held by researchers standing on top of the paver deck. The paver was oriented so that each paver profile (front, back, left-side, right-side) faced into the wind for three tests. The sequence of orientations were randomized in blocks of four. Only the 1.5 hp fan was tested outdoors. The outdoor evaluations revealed an overall average capture efficiency of 68 percent. Compared to the indoor evaluation, the outdoor results showed a 27 percentage point decline in capture efficiency and increased variation in results as wind gusts hampered the control's ability to consistently capture the surrogate contaminant. The outdoor exhaust flow rate averaged 1,370 cfm.

TABLE I. EXHAUST FLOW RATE TRIALS

	Q <sub>(SF4)</sub>	Q <sub>(exh)</sub> (Range)	Q <sub>(exb)</sub> (Average)
10 hp fan, Indoor 1a	0 569 lpm	1,103 - 1,116 cfm	1,111 cfm
10 hp fan, Indoor 1b	1 132 lpm	1,133 - 1,148 cfm	1,139 cfm
1 0 hp fan, Indoor 2a	0 <b>569 lpm</b>	1,090 - 1,109 cfm	1,100 cfm
1.0 hp fan, Indoor 2b*	0 569 lpm		1,103 cm
1.0 hp fan, Indoor 3a*	1.132 jpm 🐉	1,141 1,152 cfm	1.147 cfm 7 34.
1 5 hp fan, Indoor 1a	0 566 lpm	1,328 - 1,358 cfm	1,342 cfm
1 5 hp fan, Indoor 1b	1 124 lpm	1,357 - 1,367 cfm	1,360 cfm
1.5 hp fan, Outdoor 1a		1,367 - 1,384 cfm	1,375 cfm 🛴
1.5 hp fan, Outdoor 1b	1.124 lpm	1,357 - 1,367 cfm	1,361 cfm

<sup>-</sup> The annotations "a" and "b" are for different SF, flow rates during the same test run

<sup>\*</sup> Engine idle was reduced from 1675 rpms to 800 rpms for two trials

TABLE II. INDOOR CAPTURE EFFICIENCY TRIALS

	Q <sub>(sf6)</sub>	Q <sub>(exh)</sub>	η (Range)	η (Average)
10 hp fan, Indoor 1a	0 569* cfm	1,105 cfm	36 - 88 %	64 %
1 0 hp fan, Indoor 1b	1 132	1,143	54 - 105 %	72 %
1.5 hp fan, Indoor 1a	0.566* 1.124	1,342	54 - 98 % 107 % ±	82 % 75.%

<sup>-</sup> The annotations "a" and "b" are for different SF<sub>6</sub> flow rates during the same test run

TABLE III. OUTDOOR TRIALS, 1.5 hp FAN ONLY FRONT OF PAVER FACING THE WIND = ZERO DEGREES

Orientation/Run	$\mathbf{Q}_{(\mathrm{SF}6)}$	η(Range)	η(Average)	Wind
0°, Run 1: 7: 3	7.5	.57= 100 % ;	-1-83 %	, 5 - 8 mph
270°, Run 1	1 124	30 - 97 %	51 %	5 - 8
180°, Run 1	1.124	24 - 108 %	56 %	7-8
90°, Run 1	1 124	51 - 93 %	73 %	3 - 9
180°, Run 2	1.124	31 - 101 %	61%	8-12
90°, Run 2	1 124	36 - 95 %	64 %	2 - 5
0°, Run 2		خ. 68 - 101 % . خ	120 :: 88.%c122	3.48.1
270°, Run 2	1 124	29 - 75 %	57%	2 - 10
180°, Run 3	1.124	: 70 - 100 % rs.	89%	3.3.5
90°, Run 3	1 124	47 - 119 %	73 %	1 - 6
270°, Run 3	(±10124 ± 2		2.44%≝=	5 - 8
0°, Run 3	1 124	59 - 89 %	76 %	3 - 9

 $<sup>\</sup>eta = Capture\ efficiency$ 

## DISCUSSION

The control system flow rate calculations for the two methods, the  $SF_6$  dilution technique and the velocity pressure technique, where within 5 percent of one another. For the indoor evaluation of the 1 0 hp fan, there seemed to be a systematic difference in the flow rates calculated using flow of 0 6 lpm  $SF_6$  (1,105 cfm) verses a flow of 1 1 lpm  $SF_6$  (1,143 cfm). This systematic difference

<sup>\*</sup> SF6 released only on the right side of the auger area

is about 3.5 percent and is probably due to low accuracy in one of the  $SF_6$  delivery flow rate calibrations during the first day. Before testing the 1.5 hp fan, a new calibration was done for the  $SF_6$  delivery system. On the second day, the exhaust flow rate calculated for the 0.6 lpm  $SF_6$  (1,342 cfm) test run was only 1 percent less than the exhaust flow rate for the 1.1 lpm  $SF_6$  (1,360 cfm) test run. These differences are small when compared to the outdoor wind effect on the capture efficiency.

The 1 5 hp fan had a 20 percent increase in flow over the 1 0 hp fan. The larger fan also increased the system's capture efficiency by 23 percent, based on the indoor sampling. The 1 5 hp fan drew the same amount of air when tested outdoors as when tested indoors, however, the capture efficiency decreased by 27 percent. In addition, the variance of the samples increased during the outdoor tests. Achieving a high average capture efficiency and maintaining high capture efficiencies without performance levels fluctuating over a wide range is desired. Empirically, the performance can be evaluated by comparing the sampling data coefficients of variation (CV)

$$CV = \frac{Standard\ deviation}{Mean} \ X \ 100$$

Controls with smaller CVs are less influenced by the environmental factors and maintained a more consistent capture efficiency. For example, the CVs obtained during indoor testing of the 1.5 hp fan were all less than 20 percent as compared to several CVs greater than 50 percent obtained while testing outdoors. The CVs for each set of data are shown in Appendix B.

## CONCLUSIONS, DISCUSSION, AND RECOMMENDATIONS

Based on the evaluation results of this report, the Caterpillar control design, when paired with the larger 1.5 hp fan, has a reasonable potential to significantly reduce worker exposure. The wind speed, asphalt fume emission rate, work habits of individuals, and other factors will effect the actual reductions in worker exposure. For example, if the wind speed is very high (15 mph range), asphalt emissions may be naturally removed from the auger area, reducing the relative effectiveness of the control system. On the other hand, if the wind speed is very low (<1 mph), the wind may not remove a significant amount of asphalt emissions from the auger area. In the low wind case, the ventilation system is expected (based on indoor testing where the wind was minimal) to remove a large percentage of the asphalt emissions, thus, the relative effectiveness of the control system will be high

Some general recommendations for further improvements to the design follow. The evaluated Caterpillar local exhaust ventilation system included enclosure, hood design, and mechanical exhaust. The enclosure covered about 60 percent of the area over the augers. Caterpillar engineers expressed concerned that covering any more of this area would obstruct the view of the operator and hamper production. Any additional enclosure techniques, especially above the ends

of the auger and the screed extension areas, could increase capture efficiency, increase resistance to cross-draft disturbances, and reduce worker exposure. However, user acceptance must still be a consideration. If the auger area cannot be enclosed any further, then improvements to the hood design and an increase in the exhaust flow rate could be made.

The hood design, including the duct to hood transition and the duct to fan transition, required improvement. Although difficult to measure on this system, significant pressure losses were expected at the hood-to-duct and the duct-to-fan transitions. Smooth (gradual) transition at these transitions would increase the exhaust flow rate of the system. In addition, the short duct height also contributed to increased pressure loss due to the large surface area to cross-sectional area ratio. Re-sizing this duct could reduce frictional losses and increase the exhaust flow rate of the system.

With the 1.5 hp fan, the ventilation system's exhaust flow rate was 1,400 cfm and air velocity measurements taken at the face of the hood ranged from 110 to 150 fpm. The air velocities decreased quickly with distance from the face of the hood. At a minimum, given the physical properties of the asphalt fume, the vapor contaminants, and the process by which they are generated, we recommend a minimum design capture velocity of 100' per minute throughout the entire auger area. This recommendation assumes very good enclosure to minimize wind interference during paving operations. Based upon the selected hood design and the dimensions of the auger area, this velocity will be incorporated into the design calculations to determine a minimum exhaust flow rate requirement. There is some concern regarding convective currents and the generated volume of rising air induced above the hot paving process. However, adequate process enclosure plus an appropriately selected capture velocity will produce a sufficient exhaust flow rate to control and remove this convective exhaust volume. Additional information on controlling contaminants from hot processes may also be found in the ACGIH Ventilation Manual

#### **ACKNOWLEDGMENTS**

We would like to thank the Caterpillar management and staff for their gracious hospitality and assistance during our visit to the Caterpillar Paving Products facility. Their commitment to the design and implementation of engineering controls to reduce occupational exposures is an admirable pledge. We would like to thank Walt Haag for his contribution on the field survey

## APPENDIX A

## ENGINEERING CONTROLS FOR ASPHALT PAVING EQUIPMENT

PHASE ONE (LABORATORY) EVALUATION PROTOCOL

**PURPOSE** To evaluate the efficiency of ventilation engineering controls used on highway-class hot mix asphalt (HMA) pavers in an indoor stationary environment

SCOPE OF USE This test procedure was developed to aid the HMA industry in the development and evaluation of prototype ventilation engineering controls with an ultimate goal of reducing worker exposures to asphalt films. This test procedure is a first step in evaluating the capture efficiency of paver ventilation systems and is conducted in a controlled environment. The test is not meant to simulate actual paving conditions. The data generated using this test procedure have not been correlated to exposure reductions during actual paving operations.

For the laboratory evaluation, we will conduct a two-part experiment where the surrogate "contaminant" is injected into the auger region behind the tractor and in front of the screed. For part A of the evaluation, smoke from a smoke generator is the surrogate contaminant. For part B, the surrogate contaminant is sulfur hexafluoride, an inert and relatively safe (when properly used) gas, commonly used in tracer gas studies

**SAFETY** In addition to following the safety procedures established by the host facility, the following concerns should be addressed at each testing site

- 1 The discharge of the smoke generating equipment can be hot and should not be handled with unprotected hands
- 2 The host may want to contact building and local fire officials in order that the smoke generators do not set off fire sprinklers or create a false alarm
- 3 In higher concentrations, smoke generated from the smoke generators may act as an irritant. Direct inhalation of smoke from the smoke generators should be avoided
- 4 All compressed gas cylinders should be transported, handled, and stored in accordance with the safety recommendations of the Compressed Gas Association
- The Threshold Limit Value for sulfur hexafluoride is 1000 ppm. While the generated concentrations will be below this level, the concentration in the cylinder is near 100 percent. For this reason, the compressed cylinder will be maintained outdoors whenever possible. Should a regulator malfunction or some other major accidental release occur, observers should stand back and let the tank pressure come to equilibrium with the ambient environment.

<u>Laboratory Setup</u> The following laboratory setup description is based on our understanding of the facilities available at the asphalt paving manufacturing facilities participating in the study. The laboratory evaluation protocol may vary slightly from location to location depending upon the available facilities.

<u>Paver Position</u> The paving tractor, with screed attached, will be parked underneath an overhead garage door such that both the tractor exhaust and the exhaust from the engineering controls exits into the ambient air. The garage door will be lowered to rest on top of the tractor and plastic or

an alternative barrier will be applied around the perimeter of the tractor to seal the remainder of the garage door opening

Laboratory Ventilation Exhaust For this evaluation, smoke generated from Rosco Smoke Generators (Rosco, Port Chester, NY) is released into a perforated plenum and dispersed in a quasi-uniform distribution along the length of the augers. Due to interferences created by the auger's gear box, this evaluation may require a separate smoke generator and distribution plenum on each side of the auger region. Releasing theatrical smoke as a surrogate contaminant within the auger region provides excellent qualitative information concerning the engineering control's performance. Areas of diminished control performance are easily determined and minor modifications can be incorporated into the design prior to quantifying the control performance. Additionally, the theatrical smoke helps to verify the barrier integrity separating the front and rear halves of the asphalt paver. A video camera will be used to record the evaluation. The sequence from a typical test run is outlined below.

- 1 Position paving equipment within door opening and lower overhead door
- 2 Seal the remaining door opening around the tractor
- 3 Place the smoke distribution tube(s) directly underneath the auger
- 4 Connect the smoke generator(s) to the distribution tube(s)
- 5 Activate video camera, the engineering controls, and the smoke generator(s)
- 6 Inspect the separating barrier for integrity failures and correct as required
- 7 Inspect the engineering control and exhaust system for unintended leaks
- 8 Deactivate the engineering controls for comparison purposes
- 9 Deactivate smoke generators and wait for smoke levels to subside
- 10 End the smoke test evaluation

Evaluation Part B (Tracer Gas) The tracer gas test is designed to (1) Calculate the total exhaust flow rate of the paver ventilation control system, and (2) Evaluate the effectiveness in capturing and controlling a surrogate contaminant under a "controlled" indoor conditions  $SF_6$  will be used as the surrogate contaminant

Quantify Exhaust Volume: To determine the total exhaust flow rate of the engineering control, a known quantity of sulfur hexafluoride (SF<sub>6</sub>) is released directly into the engineering control's exhaust hood, thus creating a 100 percent capture condition. The SF<sub>6</sub> release is controlled by two Tylan Mass Flow controllers (Tylan, Inc., San Diego, CA). Initially, the test will be performed with using a single flow controller calibrated at 0.35 lpm. A hole drilled into the engineering control's exhaust duct allows access for a multi-point monitoring wand into the exhaust stream. The monitoring wand is oriented such that the perforations are perpendicular to the moving air stream. A sample tube connects the wand to a Bruel & Kjaer (B&K) Model 1302 Photo acoustic Infra-red Multi-gas Monitor (California Analytical Instruments, Inc., Orange, CA) positioned on the exterior side of the overhead door. The gas monitor analyzes the air sample and records the concentration of SF<sub>6</sub> within the exhaust stream. The B&K 1302 will be programmed to repeat this analysis approximately once every 30 seconds. Monitoring will continue until we

approximate steady-state conditions are achieved. The mean concentration of  $SF_6$  measured in the exhaust stream will be used to calculate the total exhaust flow rate of the engineering control. The equation for determining the exhaust flow rate is

$$Q_{(exh)} = \frac{Q_{(SP_6)}}{C_{(SP_6)}} \times 10^6$$
 Equation 1

where

 $\mathbf{Q}_{\text{(exh)}}$  = flow rate of air exhausted through the ventilation system (lpm or cfm)

 $Q_{(SF6)}$  = flow rate of SF<sub>6</sub> (lpm or cfm) introduced into the system

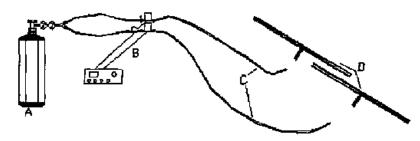
 $C^{\star}_{(SF6)}$  = concentration of  $SF_6$  (parts per million) detected in exhaust

[To convert from liters per minute (lpm) to cubic feet per minute (cfm), divide lpm by 28 3 ]

In order to increase accuracy, the exhaust flow rate will be calculated a second time using two mass flow controllers, each calibrated at approximately 0.35 lpm of  $SF_6$ . Sufficient time will be allowed between all test runs to allow area concentrations to decay below 0.1 ppm before starting subsequent test runs.

Quantitative Capture Efficiency: The test procedure to determine capture efficiency is slightly different than the exhaust volume procedure. The mass flow controllers will each be calibrated for a flow rate approximating 0.35 liters per minute (lpm) of 99.8 percent  $SF_6$ . The discharge tubes from the mass flow controllers will each feed a separate distribution plenum, one per side, within the paver's auger area. The distribution plenums are designed to distribute the  $SF_6$  in a uniform pattern along the length of the auger area. (See Figure 1.) The B&K multi-gas monitor analyzes the air sample and records the concentration of  $SF_6$  within the exhaust stream until approximate steady-state conditions develop. Once this occurs, the  $SF_6$  source will be discontinued and the decay concentration of  $SF_6$  within the exhaust stream will be monitored to indicate the extent in which general area concentrations of non-captured  $SF_6$  contributed to the concentration measured in the exhaust stream

#### FIGURE 1



#### LEGEND

A-Tracer Gas Cylinder with regulator

B-Tylon Mass Flow Controllers with Control Box

C-PIFE Distribution Tubes

D-Tracer Gas Astribution Plenurs

A capture efficiency can be calculated for the control using the following equation

$$\frac{C_{(SF_6)} \times Q_{(exh)}}{10^6}$$
Equation 2A

where

 $\eta = capture\ efficiency$ 

 $C_{(SF6)}$  = concentration of  $SF_{\delta}$  (parts per million) detected in exhaust

 $\mathbf{Q}_{(exh)}$  = flow rate of air exhausted through the ventilation system (lpm or cfm)

 $\mathbf{Q}_{(SF6)}$  = flow rate of  $SF_6$  (lpm or cfm) introduced into the system

[To convert from liters per minute (lpm) to cubic feet per minute (cfm), divide lpm by 28 3 ]

**NOTE** When the flow rate of  $SF_6[Q_{(SF_6)}]$  used to determine the engineering control's capture efficiency is the same as that used to quantify the exhaust flow rate, equation 2A may be simplified to

$$\eta = \frac{C_{(SP_6)}}{C_{(SP_6)}^*} \times 100$$
 Equation 2B

where the definitions for  $C^*_{(SF6)}$ ,  $\eta$ , and  $C_{(SF6)}$  remain the same as in equations 1 and 2A

The sequence from a typical test run is outlined below

- 1 Position paying equipment and seal openings as outlined above
- 2 Calibrate (outdoors) both mass flow meters at approximately 0.35 lpm of SF<sub>6</sub>
- 3 Drill an access hole in the engineering control's exhaust duct on the outdoor side of the overhead door, and position the sampling wand into the hole
- While maintaining the SF<sub>6</sub> tanks outdoors, run the discharge hoses from the mass flow meters to well-within the exhaust hood(s) to create 100 percent capture conditions
- With the engineering controls activated, begin monitoring with the B&K 1302 to determine background interference levels
- 6 Initiate flow of SF<sub>6</sub> through a single mass flow meter
- 7 Continue monitoring with the B&K for five minutes or until three repetitive readings are recorded
- 8 Deactivate flow of the SF<sub>6</sub> and calculate exhaust flow rate using the calculation identified above
- 9 Repeat steps #2 through #8 using both mass flow controllers
- Allow engineering control exhaust system to continue running until SF<sub>6</sub> has ceased leaking from the discharge hoses then remove the hoses from the hoods
- 11 End the exhaust flow rate test
- 12 Locate an SF<sub>6</sub> distribution plenum on each side of the auger area and connect each plenum to the discharge hose of a mass flow meter
- Initiate B&K monitoring to establish background interference levels until levels reach 0.1 ppm or below
- 14 Initiate SF<sub>6</sub> flow through the mass flow meters and monitor with the B&K until approximate steady-state conditions appear
- 15 Once steady-state is achieved, discontinue SF<sub>6</sub> flow and quickly remove the distribution plenums and discharge hoses from the auger area
- 16 Continue monitoring with the B&K to determine the general area concentration of SF<sub>6</sub> which escaped auger area into the laboratory area
- 17 Discontinue B&K monitoring when concentration decay is complete
- 18 Calculate the capture efficiency
- 19 Repeat steps 11 18 as time permits

## APPENDIX B

## ENGINEERING CONTROLS FOR ASPHALT PAVING EQUIPMENT

## TRACER GAS EVALUATION RESULTS B&K DATA FILES AND CALCULATION RESULTS

Barber-Greene	(CAT)	DeKalb.	Illinois	3/12-	15/1996	
<u> </u>	<u>,                                     </u>	Summar				
<del></del>	<u> </u>	SUITINE	y lable	<del></del>		
INDOOR, SMALL FA	! B.I	<del> </del>			<u> </u>	
				Range		<u> </u>
Flow rate #1	1111		1103	to	1116 cfm	<del></del>
Flow rate #2	1139		1133		1148 cfm	<u> </u>
Flow rate #3	1100		1090	to	1109 cfm	<u> </u>
Flow rate #4 *	1103		1096		1109 cfm	<u> </u>
Flow rate #5 *	1147		1141		1152 cfm	<u> </u>
i-Engine id	e was redu	ced from 16	75 rpm to 8	00 tbw	<u> </u>	·
<u> </u>	<u> </u>	· • — — — — — — — — — — — — — — — — — — —				<u> </u>
Capture efficiency, RI		64%			88%	<u> </u>
Capture efficiency, Fu	HI	72%	54%	to	105%	<del></del>
	<u> </u>	1	<u> </u>			<del></del>
INDOOR, LARGE FA						<u> </u>
Flow rate #1	1342	4	1328		1358 cfm	<u> </u>
Flow rate #2	1360	¢fm	1357	to	1367 cfm	
	<u> </u>				!	
Capture efficiency, Rt		82%	<del></del>	to	98%	<u> </u>
Capture efficiency, Fu	ifl	95%	74%	to	107%	1
					]	
OUTDOOR, LARGE				L		1
Flow rate #1	1375	cfm	1367	to	1384 cfm	1
Flow rate #2	1361	cím	1357	to	1367 cfm	
	1	,	· ·		, , ,	Wind Speed
OUTDOOR, LARGE	FAN, WIND	FROM FR	ONT TO BA	ACK OF	PAVER	mph
Capture efficiency, Rt		83%		to	107%	5-7
Capture efficiency, Fu		83%			100%	
Capture efficiency, Rt		75%		to	92%	3 - B
Capture efficiency, Fu		88%			. 101%	<del></del>
Capture efficiency, Rt		81%			86%	3 - 8
Capture efficiency, Fu		76%			89%	
,	,				1	·
OUTDOOR, LARGE	FAN. WIND	FROM RIG	HT TO LE	FT OF	PAVER	
Capture efficiency, Rt		55%			92%	5 -B
Capture efficiency, Fu		51%			97%	<del>-                                    </del>
Capture efficiency, Ri		76%			97%	2 - 5
Capture efficiency, Fu		57%			75%	
Capture efficiency, Rt		65%			86%	1-6
Capture efficiency, Fu		44%			72%	; 1- <u>0</u>
Capitale emplemby, Ft	111, TO	****	2170	. 10	* FW1	·- <u>-</u> ·- ·
OUTDOOR, LARGE	EAN MUNIC	EDAM DA	CK TO ED	NY AS	DAVED	<del></del>
		63%			118%	7 - 8
Capture efficiency, RI			<del></del>		<del></del>	· · · · · · · · · · · · · · · · · · ·
Capture efficiency, Fu		56%			108%	<u> </u>
Capture efficiency, Rt		69%	<del></del>		108%	8 - 12
Capture efficiency, Fi		61%	<del></del>	+	101%:	· 
	Oniy, #3`	90%			113%	3 - 5
Capture efficiency, Rt		. en.0/	70%	; to	100%	<u>†</u>
Capture efficiency, Ri		89%	<del>+</del>			<del></del>
Capture efficiency, Fi	ıli, #3					
Capture efficiency, Fu	ili, #3 I FAN, WIND	FROM LE			PAVER	
Capture efficiency, Fi OUTDOOR, LARGE Capture efficiency, Re	ili, #3 FAN, WIND only, #1	FROM LE 65%	29%	· to	PAVER 102%	3.9
OUTDOOR, LARGE Capture efficiency, Ri Capture efficiency, Ri	ili, #3     FAN, WIND   only, #1   ill, #1	FROM LE 65% 73%	29% 51%	to to	PAVER 102% 93%	
Capture efficiency, Fi OUTDOOR, LARGE Capture efficiency, Re	ili, #3     FAN, WIND   only, #1   ill, #1	FROM LE 65%	29% 51%	to to	PAVER 102%	3.9
OUTDOOR, LARGE Capture efficiency, Ri Capture efficiency, Ri	ili, #3     FAN, WINC   only, #1   ill, #1   only, #2	FROM LE 65% 73%	29% 51% 40%	to to	PAVER 102% 93%	
Capture efficiency, Fil OUTDOOR, LARGE Capture efficiency, Ri Capture efficiency, Fil Capture efficiency, Ri	ili, #3 	FROM LE 65% 73% 67%	29% 51% 40% 38%	to to to to	PAVER 102% 93% 143%	
Capture efficiency, Figure 2007 DOOR, LARGE Capture efficiency, Figure efficiency, Ricapture efficiency, Figure efficiency, Fig	ili, #3 	55% 73% 57% 64%	29% 51% 40% 36% 48%	to to 10 to	PAVER 102% 93% 143% 95%	2 - 10

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1302 Settings	<del></del> ,		· · · <del></del>			
1			· ·	<del> </del>		
Compensate for Water Va	an Interference		NO.	<del> </del> <del>-</del>		
Compensate for Cross Inf	eference	NC		<del></del>	<del> </del>	
Sample Continuously	CHECKIDE	YES	,	<del></del>		<del></del>
Pre-set Monitoring Period		NO		<del>'</del>	<del></del> <del></del>	<del> </del>
		<del></del>		<del></del>		
Measure		<del></del> +		1		
Gas A Formaldehyde	-	NO		<del></del>		
Gas B Carbon dioxide		· - · - · -		;	- 1	
Gas C Carbon monoxide		NO	· · · ·	<u> </u>		
Gas D TOC as Propane		NO				
Gas E Sulfur hexafluorid	e	YES		1	1	
Water Vapour		NO		1	:	
Sampling Tube Length		15 0 ft				
Air Pressure		00 mmHg				
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General Information				<u>.                                      </u>	!	
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15 11 48 12	3 35E-02	0 038521	<del> ,</del>			
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27 11 55 29	2 52E-D2	0 02983				
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39 12 03 10	1 69E+01			· <del></del>	···	<u></u>
40 12 03 45	1 69E+01					<del></del>
41 12 04 21	1 68E+01					<u>!                                    </u>
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49 12 09 04	3 21E+01			<del></del>		<u> </u>
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52 12 10 50	3 22E+01				1148 133	
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61 12 16 33	3 90E-02		Std Dev	0 004924		
62 12 17 09	5 07E-02	0 056531	CV	10 21%		
12 17 44 User I	7	'		1		
63 12 17 44	4 D2E-02	0 045537				
64 12 18 20	1 71E+01			- ;		
12 19 00 User	8	_				
65 12 19 00	1 70E+01	18 21419	Rt side on	ly SF6 1009	& capture	
66 12 19 35	1 70E+01			@ 1675 rpr		
67 12 20 11	1 69E+01			. 18 25807		Mean flow
68 12 20 46	1 71E+01			0 125047		
69 12 21 33	1 72E+01			0 68%		
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70 12 22 08		18 32386	Rt side on	ly SF6 100	4 centure	
71 12 22 44	1 69£+01			@ 800 rpm		<u> </u>
72 12 23 20	1 70E+01				<u> </u>	
73 12 23 55	1 70E+D1			<del></del> -	<del></del>	·
74 12 24 30	1 69E+01			·		<del>                                       </del>
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80 12 28 03	3 20E+01			1		·
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84 12 30 25	3 17E+D1			F6 into dist	ribution te	es
85 12 31 00		0 184619			<u> </u>	
86 12 32 12	5 13E-02			<u>i</u>		!
87 12 32 47	4 85E-02	0 054228				,
88 12 33 23	3 17E-D2	0 036536				
89 12 33 58	3 12E-02	0 036112				
90 12 34 33		0 D33076				i
91 12 35 09	3 28E-DZ	D 03778B		-		
92 12 35 44		0 035275		1 .		1
93 12 36 20	, 2 16E-02			<del> </del>		1
12 36 55 User	12					
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95 12 37 31	1 38E+01					
96 12 38 09	1 13E+01	12 0137				
97 12 38 44	1 11E+01	11 79793		<del></del>		<del>*</del>
98 12 39 19	1 42E+01			1 ,		·
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100 12 40 30	9 51E+00			<del>,</del>	<del></del>	<u> </u>
101 12 41 25	8 43E+00					·
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106	12 44 22	1 50E+01	16 02727		3 032574		
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111	12 47 22	2 65E+01	28 77657	<del></del>	<del></del>		<del>}</del> _
112	12 47 57.			<del></del>			<u> </u>
113	12 48 32	1 85E+01	20 19311		<del></del>		<u> </u>
114	12 49 08	1 93E+01	20 74462	!			
115	12 49 43	2 43E+01			<del></del>	<del></del>	<u> </u>
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117	12 50 54	2 64E+D1	28 66384	<del></del>	<del></del>		:
118	12 51 41	3 35E+D1			25 16977	72.00%	Ave Eff
119	12 52 16	1 75E+01		Sid Dev	5 733306		Min Eff
120	12 52 52	2 63E+01			22 78%	105 13%	
121	12 53 27	2 53E+00			22 10/8	100 1070	14107 1-11
122	12 54 05	2 39E+00	2 513632		<del>`</del>		<del>:</del> -
12 54 41 6		14	2013032	2			<del>!</del> -
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127	12 57 05	4 01E-01			·		;
128	12 57 40		0 354348			<del></del>	<del></del>
129	12 58 16		0 452865				<u>-                                      </u>
130	12 58 51		0 396267				
131	12 59 26	4 17E-01	0 440287				
132	13 00 02	1 69E-01	0 180429			•	;
133	13 00 37	2 73E-01	0 289381				1
134	13 01 44	1 85E-D1	0 197189	· · · · · · · · · · · · · · · · · · ·	<u> </u>		<u> </u>
135	13 02 19	2 56E-D1	0 27157				1
136	13 02 55		0 177287				<del>****</del>
137	13 03 30	2 06E-01	0 219187				<del>:</del>
138	13 04 D5		0 094545				<del>-</del> -
139			0 118633				
13 05 16 (		15					i
140	13 05 16		0 277856	Backgrou	nd, inside c	arage	1
141	13 05 51			Some Ros			1
142	13 06 27			generated			1
143	13 07 D2		0 101457		1		1
144	13 07 38,		0 085015		<del> </del>		<del>                                     </del>
145			0 072867				1
146	13 08 48		0 095488				1
147	13 09 24		0 099153				1
148	13 09 59	9 88E-02			<del></del>	<del></del>	<del>:                                    </del>
149	13 10 34		0 075904				T
150	13 11 29		0 103656			<del></del>	
151	13 12 05		0 093707				<del></del>
					·		

CAT, DeKalb, Illino	is 3/12-15/1990	6	1 .		<del></del>
Large Fan	···	ide, engine	outside	i	
1202 Management Date	1804003/0003	4000 00 44 46	36 8 4		
<ul> <li>1302 Measurement Data</li> <li>1302 Settings</li> </ul>	18D4892/2803	- 1995-03-14 16	26 - Page 1	<u>•</u>	
TOOL Senings			<del>:</del>		
	!		<u> </u>		
Compensate for Water Va	o Interference :	NO	<del>{</del>	<del></del>	
Compensate for Cross Inte		NO	<del> </del>		
Sample Continuously	YE		<del>-</del>	i	<del></del>
Pre-set Monitoring Period		io —	<del>}</del>	<del></del>	
			<del></del>		
Measure		<del>- •</del>	:		
Gas A Formaldehyde	<u> </u>	10	!	<del></del>	
Gas B Carbon dioxide		10	•		
Gas C Carbon monoxide		NO	·		
Gas D TOC as Propane		NO	· · · ·		
Gas E Sulfur hexafluoride	Υ Υ	ES	<del>, -</del>		
Water Vapour	NO				·
Sampling Tube Length	15	011			
Air Pressure	759 0 m	mHg			<u> </u>
Normalization Temperatur	e	500 F			
General Information			·		
Start Time	1995-03-14 09				
Stop Time	1996-03-14 10	22	· <u> </u>		· 
Results Not Averaged		<del></del> .	1		· <del>· · · · · · · · · · · · · · · · ·</del>
Number of Event Marks		5	<del>-,</del>		
Number of Recorded Sam	pies	56	·		
		· <del></del>	<del> </del>	<del></del>	
Samples Measured From	1996-03-14 09 47		<del></del>	·	
	P				
Samp Time No hhmmiss Event	Response Calit			<del></del>	
lo hhmm ss Event	PPM COM	ection			·
1 9 47 48	3.40E-02. 0.0	39044 Backgrou	nd in sybo	uet etack	
1 9 47 48 2 9 48 31:		03517	11d, III 6X116	Mar Plack	
3 9 49 07	2 81E-02 0.0		<del></del>	SF6 flow	
4 9 50 01	3 24E-02 D 0		<del></del>	Rt side	#169
5 9 50 37	3 29E-02 D 0		ļ	0 5662	logg
6 951 12	2 93E-02 O 0		<u> </u>	Both sides	
7 95148		04962	<del>-</del>	1 1235	
8 9 52 23	6 50E-02 0.0		<del></del>	1 1200	·piii
9 9 52 58		01379 Avg	0 063435		<del></del>
10 9 53 34		81664 Std Dev	· 0 051267	· · · · · · · · · · · · · · · · · · ·	<del> </del>
11 9 54 09		52762 CV	80 82%		<del>!</del>
9 54 45 User	1	<u></u>		<del> </del>	·
12 9 54 45		51237 Rt side o	niv SEG 100	% capture	
15 504 40		, DING 0.	, 100		

## CAT\_Inside\_Lg

<del></del>							
13	9 55 23	1 40E+01	14 93843		[ <u> </u>		
14	9 55 58	1 39E+01	14 82871		I"t		
15	9 56 33	1 38E+01	14 72103		1		
16	9 57 09	1 40E+01	14.93843		,		
17	9 57 44	1 40E+01	14 93843		, -		
18	9 58 20	1 40E+01	14.93843	Avq	14 89767	1341 61	Mean flow
19	9 58 55	1 41E+01			0 099596	1328 28	Min
20	9 59 30	1 39E+01			0 67%	1357 708	Max
10 00 17 1	User	2					
21	10 00 17	2 70E+01	29 34076	Both side:	SF6 100%	capture	
22	10 00 55	2 67E+01			1		
23	10 01 30	2 69E+01			<del>'</del>		
24	10 02 05	2 69E+01			<del> -</del>		<del></del>
25	10 02 41	2 69E+01					
26	10 03 16	2 68E+01			29 16336	1359 91	Mean flow
27	10 03 52	2 68E+D1		Std Dev	0 088796		<u> </u>
2B	10 04 27	2 69E+D1			0 30%	1367 47	
10 05 03 1		3		<del></del>			
29	10 05 03		D 145864	Placing S	F6 into dist	ribution te	es
30	10 05 43		0 056322				
31	10 05 18		0 048259		<del></del>		<u>.                                    </u>
10 06 54 1		4					<del></del>
32	10 06 54	1 53E-D1	0.16367	Rt side or	ly, distribu	tion	
33	10 07 29	1 24E+01			;		·
34	10 08 07	1 37E+01			<del>.</del>	· ———	<del></del> -
35	10 08 43	1 05E+01		<del></del>	<del></del>		•
36	10 09 18	7 60E+00		<del></del>			
37	10 10 24	1 34E+01				<del></del>	<u>·</u>
38	10 11 00	1 37E+01			<del></del>		<del>;</del>
<del>39</del>	10 11 35	9 07E+00			12 19602	81 97%	Ave Eff
40	10 12 11	1 11E+01			2 299524		'Min Eff
41	10 12 46	1 17E+01			18 85%		'Max Eff
42	10 13 22	1 48E+01			10 03%	80 U0%	MINY EII
42	10 13 57	1 61E+01			<del></del>		
		5	11 44004	<del></del> .	<del></del>		<u> </u>
10 14 33			40 22200	Dath are	<u> </u>	L	<del>}</del> -
44	10 14 33				s, distribut	iun	
45	10 15 10	2 83E+01			<u> </u>		<u> </u>
46	10 15 46	2 87E+01			<u> </u>		<del>!</del>
47	10 16 21	2 70E+D1			·		
48	10 16 57.	2 59E+01			<del></del>		<del> </del>
49	10 17 32	2 39E+01			<u> </u>	·	<del></del> -
50.	10 18 08	2 01E+01			1		<del>'</del> -
51_	10 18 43	2 44E+01			·		
52	10 19 18	2 57E+01			27 7628		Ave Eff
53	10 20 13,				3 010658		Min Eff
54	10 20 49	2 83E+01			10 84%	107 21%	Max Eff
55			0 577619			<u> </u>	<u> </u>
56	10 22 05	6 05E-01	0 637388		•		

CAT. >Katb, Illinois	3/12-15/1996	i	
Large ian	Outside testing		
	- 1804892/2803 - 1996-03-14 16 24 - Page 1		
1302 Settings	<u> </u>		
<u> </u>	i !		
Compensate for Water Vap In			
Compensate for Cross Interfer		- <del></del>	
Sample Continuously	YES	<del></del>	
Pre-set Monitoring Period	NO	<del></del>	
<del></del>	- <del> </del>		
Measure	<u></u>	·——	
Gas A Formaldehyde	NO	<del></del>	
Gas B Carbon dioxide	NO	<del></del>	··· — —
Gas C Carbon monoxide	NO	· · · · · · · · · · · · · · · · · · ·	
Gas D TOC as Propane	NO	<del> </del>	
Gas E Sulfur hexafluonde	YES		· <del>-</del> '
Water Vapour	NO .		
Sanahar Taka Larath	45.06	<del></del>	
Sampling Tube Length Air Pressure	15 0 ft		
	759 0 mmHg		
Normalization Temperature	50 0 F	<u> </u>	
General Information		<del></del>	
General information		·	<del></del>
Start Time	1995-03-14 10 58	······································	
Stop Time	1996-03-14 10 98	<u></u>	
Results Not Averaged	1999-03-14 12 00	<u>'</u> -	
Number of Event Marks	<del> 7</del>	<u> </u>	<u></u> <u></u>
Number of Recorded Samples	106	<u>!</u>	
Transper of Recorded Samples			
Samples Measured From 1996	02.14.10.68		
Samples Measured From 1990	-03-14 10 50		·
Samp Time	Response Calibration	<del>;</del>	
No hhmmss Event	PPM Correction	<del>'                                    </del>	
THE THIRT'S EVENT	TT (8) CONTECTION	<del>,                                    </del>	
1 10 58 42 3 675 02	0 041872 Wind blowing front to back, 0 o	ionroec	<del></del>
	0 044908 Wind speed at about 6 mph	i cathon	<del></del>
3 11 00 01 3 03E-02		<del> </del>	
4 11 00 36 2 76E-02		<del> </del>	<del></del>
5 11 01 12 2 69E-02		SF6 flow to	ofac
	0 036112	Both sides	
7 11 02 23 3 03E-02		1 1235	lom
8 11 02 58 3 05E-02		1 1233	192141
9 11 03 33 2 97E-02		<del>! ;</del>	<del></del>
10 11 04 09 2 81E-02		<del> </del>	
		<del></del>	
		·	
		<u> </u>	
13 11 05 57 3 54E+D0	3 727087		

## Outside1\_Lg\_Fan

14 11 06	ŝ <u></u>	11 15131	<del></del> _				
15 11 07	<u>3⊑ 11</u> 3€+01	12 66175	<del>_</del>				<del>!</del> — — —
16 11 07	1E+01	12 878	<del></del>	<del> :</del>			<del></del>
17 11 08	4E+01			<u></u>			<u> </u>
<del></del>	:2E+01	13 20261	<del></del>	<del></del>	<del></del>	<del></del>	
	16E+01	12 98617	<del></del>		<del></del>	<del></del>	<del></del>
<del></del>		15 59136				4	<del></del> -
	)2E+00 15E+01	10,52726		12 41263			<del> </del>
		15 48246		1 930439			<u> </u>
22 11 11 27 11 12 02 User	-9E+01	15 91825	CV	15 55%	106 83%	Max EII	<del></del>
23 11 12 02	12E+01	14.0058	D-45 - 3 - 5				<del> </del>
24 11 12 33	34E+01	20 85502	Both sides		on	·	1
	711E+D1	20 65502				·	<del></del>
	2 65E+01	28 77657	<u></u>	<del></del> ;			<del></del>
	2 27E+01	24 51625				<del></del>	<del></del>
	2 23E+01	24 07058		<del></del>		L	
	2 56E+01	27 76318					
	2 40E+01	25 96831		- · <del></del>			<del></del>
	1 53E+01	16 35452	Δυρ	23 82946	B2 64 0/.	Ave Eff	<u> </u>
	2 28E+01		Std Dev	3 544242		Min Eff	
<del> </del>	2 10E+01	22 62576		14 87%		Max Eff	1
	5 95E+00			14 07 76	401170	INIAX EII	1
11 19 48 User	2 =05 +00	0 202/42	····	· •		<del></del>	<del></del>
L •	4 43E-01	0.46754	Wind blow	on right to	N toff that of	007406	
	4 55E-02	D 051086	Wind spee	d at about	6 to 7 mm	- Cyree	<del></del>
	3 50E-02		Mind Phes	u al about	o to vinipi	<u>'</u>	
38 11 21 37		0 067213	<del></del>	<del></del>		<del></del>	
	5 16E-02			_			
	3 14E-02	0 036322		<del></del>	<del></del>		
41 11 23 24		0 047002			<del></del>		
42 11 23 59	4 09E-02	0 04527	<del></del>	<del></del>			···
11 24 34 User	3		<del></del> -				
		0.032552	Rt side on	u dietribi	tion		<del>,</del>
	2 60E+00			,,		<del></del>	,
	3 93E+00					<del></del>	· · · · · · · · · · · · · · · · · · ·
	7 37E+DD						!
	1 29E+01						1
	6 68E+00						
	7 62E+00			8 13828	54 62%	Ave Eff	
	7 44E+00			2 851964		Min Eff	<del></del>
	7 84E+00			35 04%		Max Eff	<del></del>
11 30 15 User	4	4-2-41	<del></del>			1	<del></del>
	8 10E+00	8 575448	Both sides	. distributi	ion	<del> </del>	<del>-                                    </del>
	4 92E+00				)		<del>-</del>
	9 35E+00				·	i	<del> </del>
<u> </u>	B 07E+00		<del></del> :			<del>;</del>	<del></del>
	9 45E+00			····		<del>}</del>	<del></del>
	1 30E+01				L	<del></del>	
	1 44E+01					<del>'</del>	
<del></del>	1 14E+D1				<del></del>	<del></del>	
	1 05E+01			14 72143	51 04%	Ave Eff	
		1 1 2 1 2 1	FTEM	1- 1- 1-D	J	- YE MI	-

## - 1\_-3\_ Outside1\_Lg\_Fan

61 11 35,34° 2 58E+01	2: 8814:6td Dev 6 682565 29 62% Min Eff
62 11°36 12 2 18E+01	21: 11423 CV 45 39% 97 03%, Max Eff
63 11 36 47 3 23E-01	
64 11 37 27 1 02E-01	
65 11 38 03 4 D8E-02	
66 11 38 49 3 47E-02	
67 11 39 24 3 00E-02	0.034856 Wind blowing back to front, 150 degrees
68 11 40 00 2 05E-02	0 024909 Wind speed at about 6 to 7 mph
69 11 40 35 3 62E-02	
70 11 41 11 2 B1E-02	
11 41 45 User 5	
71 11 41 46 2 45E-01	0 260045 Rt side only, distribution
72 11 42 21 8 11E+00	
73 11 42 59 5 59E+00	
74 11 43 35 1 64E+01	
75 11 44 10 8 29E+00	
76 11 44 45 6 22E+00	
77 11 45 21 7 34E+00	
78 11 45 57 6 92E+00	
	11 79793 Std Dev 3.779396 39 59% Min Eff
	9 207119 CV 40 37% 117 83% Max Eff
11 47 45 User 6	
	13 96108 Both sides, distribution
82 11 48 23 6 67E+DD	
83 11 49 30 8 53E+00	
84 11 50 05 8 73E+00	
85 11 50 41 7 90E+00	
86 11 51 16 1 96E+D1	
87 11 51 54 2 49E+D1	26 97687 Avg 16 15689 55 01% Ave Eff
88 11 52 29 1 53E+01	16 35452 Std Dev 9 322035 24 44% Min Eff
89 11 53 07 2 86E+01	31 1519 CV 57 70% 108 00% Max Eff
11 53 45 User 7	
	35 37435 Both sides SF6 100% capture
91 11 54 21 2 19E+01	
92 11 54 56 2 50E+O1	
93 11 55 31 2 78 E+01	
94 11 56 07 1 51E+01	
95 11 56 45 2 67E+01	
96 11 57 22 2 67E+01	
97 11 57 58 2 65E+01	
98 11 58 33 2 64E+01	
89, 11 59 28 2 65E+01	
	0 220235
101 12 00 44 6 09E-02	
102 12 01 19 4 11E-02	
103 12 01 55 3 40E-02	0 039044
104 12 02 30 3 49E-02	
105 12 03 06 3 97E-02	0 045013
12 05 58 1302	0.057449
106 12 05 58 4 68E-02	
1 12 07 44	3 23E-02 0 037264 Wind blowing left to right, 270 degrees

2	12 08 27	3 36E-02	oc. 🤫	W <sub>i</sub> , 1 spee	d at about	6 mph	1
3	12 09 02	3 42E-02	د0 0	Backgrou	nd, in exha	ust stack	
12 09 37		1					
4	12 09 37	2 55E-02	0 03C	Rt side on	ly, distribu	tion	
5	12 10 12	4 05E+00	4 26€				
6	12 10 50	1 42E+01	15 155				
7	12 11 26	1 08E+01	11 474				
8	12 12 01	1 04E+01	11.043u				1
9	12 12 36	8 48E+00	8 98217				•
10'	12 13 12	1 5 60E+00	5 9100/	Avg	9 678156		Ave Eff
11	12 13 47	9 96E+00	10 570_	Std Dev	3 380239		'Min Eff
12	12 14 22	9 45E+00	10 0222:	CV	34 93%	101 72%	Max Eff
12 14 58		2	4				
13	12 14 58	1 03E+01			s, distribut	on	
14	12 15 33	3 30E+01	36 17866			)	1
15	12 16 22	1 37E+01	14 61237				
16	12 17 00	2 13E+01	22 95869				
17	12 17 37	2 14E+01	23 06973				
18	12 18 13	1 59E+01	17 00985				<u> </u>
19	12 18 50	2 47E+01	26 75251		4		1
20	12 19 28	1 91E+01	20 52392	Avg	21 03742		Ave Eff
21	12 20 04	1 90E+01	20.41362		3 823434		Min Eff
22	12 20 39	2 13E+01	22 95869	CV	18 17%	92 76%	Max Eff

CAT, DeKalb, Illinois	3/12-15/1996	1	
Large fan	Outside testing		
- 1302 Measurement Data	1804892/2803 - 1996-03	-14 2-Page 1-	<del>-                                    </del>
1302 Settings	1	E-Fago I-	╅
			<del> </del>
,			
Compensate for Water Vap	Interference NO		
Compensate for Cross Inter			
Sample Continuously	YES		
Pre-set Monitoring Period	NO NO		
		!	
Measure Con A Formulation		<del></del>	<del>-</del>
Gas A Formaldehyde	NO NO	<del></del>	<del></del> -
Gas B Carbon dioxide Gas C Carbon monoxide	NO NO		
Gas D TOC as Propane	NO NO	<del></del>	
Gas E Sulfur hexafluonde	YES	<del></del>	
Water Vapour	NO NO	<u> </u>	<del></del>
,			
Sampling Tube Length	15 0 fi	<del></del>	<del></del>
Air Pressure	759 0 mmHg	<del></del>	
Normalization Temperature	50 0 ₱		· · · · · · · · · · · · · · · · · · ·
	<del></del>	<del>-</del>	
General Information			!
Start Time	1996-03-14 14 09		:
Stop Time	1996-03-14 15 10		
Results Not Averaged		<u> </u>	· · · · · · · · · · · · · · · · · · ·
Number of Event Marks	10		·
Number of Recorded Samp	les <u>98</u>		
Caroleo Managed English	100 02 14 14 10	<del> </del>	<del></del>
Samples Measured From 1	980-03-14-14-10	<del></del>	
Samp Time	Response Calibration		<del></del>
No hh mm ss Event	PPM Correction		<del></del>
NO MITHIUS EVEN	Prior Correction		<del></del>
12 21 14 User	3		<del></del>
23 12 21 14		nd blowing back to front,	180 deprees
24 12 21 55		nd speed at about 9 mph	<u></u>
25 12 22 30	· 3 16E-02 D 036531		-
26 12 23 06	3 16E-02 0 036531		
27 12 23 41	4 40E-02 D 049516		1 -
28 12 24 16	4 68E-02 D 052448		
29 12 24 52	1 16E+00 1 21982		
12 25 27 User	4		1
30 12 25 27		side only, distribution	1
31 12 26 03	7 08E+00 7.485786	1	
32 12 27 11 33 12 27 47	1.51E+D1 16 13632 1 58E+D1 16 90055		

## Outside2\_Lg\_Fan

<u> </u>	0.055.00	0.50	<del></del>		<del></del>
34 12 28 22	9 05E+00			<del></del>	
35 12 28 58	8 98E+DD	· —		60.0400	- <del></del>
36 12 29 33	4 20E+00		10 28636		
37 12 30 08	1 13E+D1	12 0137 Sid De		29 70%	
36 12 30 44		6 218295 CV	43 60%	108 30%	Wax Fil
12 31 19 User	5				<del></del>
39, 12 31 19		8 596844 Both s	ides, distributi	on	<u> </u>
40 12 31 55	8 50E+00				<u> </u>
41. 12 32 30	. 1 78E+01	19 09245			
42 12 33 06	8 75E+00		· · · · · · · · · · · · · · · · · · ·		
43 12 33 41.	1 93E+01				
44 12 34 19	1 91E+01		1		
45 12 34 54,		13 09438 Avg	17 46514		Ave Eff
46 12 35 32	1 75E+01				Min Eff
47 12 36 08	2 69E+01	29 22785 CV	38 75%	101 34%	Max Eff
			<del>-</del> -		
1 14 10 04		0 044385 Wind b			degrees
2 14 10 47		0 036217 Wind s			<u> </u>
3 14 11 23		0 030772 Backg	round, in exha		
4 14 11 58	2 86E-02	0 03339			I
14 12 33 User	<u>1</u>				
5 14 12 33	3 20E+00	3 367943 Rt side	only, distribu	tion	
6 14 13 11	8 50E+00	9 003589	1		4
7 14 13 47	1 36E+D1	14.50375			!
8 14 14 22	5 67E+00	5 984413			,
9 14 14 57	5 69E+00	6 005669			·
10 14 15 33	7 45E+00	7 880708			
11 14 16 08	7 03E+00	7.432448 Avg	10 0279	67 30%	Ave Eff
12 14 16 43	1 98E+01	21 29694 Std De	v 5 286167	40 16%	Min Eff
13 14 17 21	7 67E+00	8 115714 CV	52 71%	142 93%	'Max Eff
14 17 59 User	<u> </u>	<del></del>			,
14 14 17 59	4 37E+00	4 605181 Both s	ides, distributi	on	1
15 14 18 34	1 32E+01				
16 14 19 10	, 2 48E+01		<del></del>	<u> </u>	<del></del>
17 14 19 48	1 31E+D1	13 96108	<del></del>		† <del></del>
18 14 20 37	9 91E+00		,		<del>:</del>
19 14 21 12		14 28659	<del></del>		,
20 14 21 47		13 85263 Avg	18 52231	63 56%	Ave Eff
21 14 22 22		27 53635 Std De			Min Eff
22 14 23 00	2 50E+01		39 18%		Max Eff
14 23 36 User	3	<u></u>	1		
23 14 23 36		15 80925 Wind (	olowing front t	o back. O	legrees
24 14 24 14		1 419405 Wind (			<u> </u>
25 14 24 52		0 041767	1		<del>1</del>
26 14 25 27	. 4 65E-02			<u> </u>	<del></del>
27 14 26 03	9 45E-02		<del>- i</del>	└╌ <u>┈</u> ───- ᡶ	<del> </del>
28 14 26 38	4 49E-02			<del></del>	<del> </del>
29 14 27 13		0.05716		<u> </u>	<del> </del>
		0 042186	<del></del> -	·	<del></del>
30 14 27 49		<del></del>	<del>-  -</del>	<u> </u>	<del>:</del>
31 14 28 24 32 14 29 00 .	1 27E-01	0 030039	<u> </u>	1	<u> </u>
32 14 29 00 .	2 54E-U2	ひいろひいろろ			•

77 44 70 07						
33 14 29 35		0 027422		<del></del>		
34 14 30 10	8 48E-02	0 092241	i	<u> </u>		
35 14 31 17	1 81E-01	0 192999	<del>' :</del>			
14 31 52 User	4 1		<u> </u>		<u> </u>	
36 14 31 52	6 27E+00		Rt side on	ly, alstribu	TION	
37. 14 32 30	1 29E+01	13 74422	<del>.</del> ļ			
38 14 33 06	8 60E+00	9 110697				
39 14 33 41	1 12E+01	11.9058				<del></del>
40 14 34 16	1 16E+01	12 33759				<del></del>
41 14 34 52	8 49E+00	8 99288	<del></del>	44 00044	75 420	- E#
42 14 35 27	1 13E+01	12 0137		11 23911		Ave Eff
43 14 36 02	9 84E+00			1 636853	60 35%	
44 14 36 38	1 07E+01	11 36673	<u>CV</u>	14 56%	92 24%	Max Eff
14 37 13 User	5	4 5 5 5 5 6 4 4 4	F - 45 - 25 - 2			·
45 14 37 13 .	5 28E+DD		Both sides	יות פחזצום ויי	ion	
46 14 37 49	2 66E+D1	28 88934				<del></del>
47 14 38 27	2 42E+D1	26 1922				<del></del>
48 14 39 02	2 72E+D1					<del></del>
49 14 39 38	2 45E+D1					l
50 14 40 13	1 85E+01			95 CD50	00 450/	A F-66
51 14 41 08	2 41E+01			25 6859		Ave Eff
52 14 41 43	2 13E+D1			3 113776		Min Eff
53 14 42 19	2 35E+01	25 40916	CV	12 12%	101 40%	Maxell
14 42 54 User	6	er teans	<b>5</b> - 45 - 2-1	055 4400		<del></del> -
54 14 42 54			Both sides	SF6 100%	capture	
55 14 43 30	2 50E+00			<u> </u>		<u> </u>
56 14 44 07	1 38E+01				CEC Flance	
57 14 44 43	1 41E+01	15 04717		<u> </u>	SF6 flow a	
58 14 45 18	1 41E+01	15 04717		<u> </u>		
59 14 45 54	2 68E+01			<u> </u>	1 1235	ipm
60 14 46 31	2 68E+D1			<del></del>		
61 14 47 07	2 69E+01				<del></del>	
62 14 47 42	2 69E+01				<u>.</u>	
63 14 48 18	2 68E+01			<del></del>	<del></del>	<del>!</del>
64 14 48 53	2 67E+01			20 12766	4304 444	Mana flavo
65 14 49 28	2 68E+01	28 11498	AVU	₹8 17130		Mean flow
66 14 50 04	2 69E+01	20 22705	Std Dev		1356 909	
67 14 50 50	<del></del>	28.22/40	<u>v</u>	D 32%	1367 47	INIAX
14 51 26 User	7.	D 1000E0			a total no d	<u> </u>
6B 14 51 26			Wind blov			
69 14 52 06			Wind spee	T AT ADOU	o to 1 mp	<del>!! - · ·</del>
70 14 52 41		D 038102		<u></u>	<del> </del>	<del> </del>
71 14 53 17		0 033599		<del></del>	<del></del>	<del>}</del>
72 14 53 52		0 034961		1	<u> </u>	<del>]</del>
73 14 54 27 .		D 034228		<u></u>	<u> </u>	<del> </del>
74 14 55 03		0 039254		<u>,                                     </u>	<u> </u>	<del> </del>
75 14 55 38		0 030668		<u> </u>	<del> </del>	<u>!</u>
14 56 14 User	<u> </u>	0.00000	Ma alda an		42	<del></del>
76 14 56 14	3 10E-02			iy, Distribi	<u>'''''</u>	<del> </del>
77 14 56 49	3 23E-01			<del> </del> -	<del> </del>	<del> </del>
78 14 57 25	, 7 86E+00	8 318766	·	<u> </u>	<u> </u>	

## Outside2\_Lg\_Fan

79	14 58	03	9	84E+DD	10 44126						
80	14 58	38	1	22E+01	12.98617						
B1	14 59	13	1	36E+01	14 50375						
82	14 59	49		24E+01	13.20261		- I				
83	15 00	55	1	06E+01	11,25901	Avg	1	11 37516	76 34%	Ave	Eff
84	<u>15 01</u>	31_	7	7.82E÷00	8 276025	Std	Dev	2.266603	55 54%	Mini	Eff_
	<u>15 02</u>	06	1	13E+01	12 0137	CV		19 93%	97 34%	Max	Eff
15 02 42 U	ser		9		_						
	15 02	42	1	25E+01	13 31087	Bott	sides :	, distribut	ion		
	15 D3	-	- 1	72E+01	18 43357		j				
88	15 03	52	1	16E+01	12.33759						
89	15 04	28	1	61E+01	17.22854						
	15 DS		7	94E+00	8 404321					<u> </u>	
	15 05			44E+01	15 3735						
92	15 06	14	2	02E+01	21 73936	Avg		16 57277	56 87%	Ave	Eff
93	15 06	52	1	.86E+01	19 97272	Std	Dev	4 384926	28 84%	Min	Eff
94	15 07	27	1	78E+01	19 09245	ÇV		26 46%	74 60%	Max	Eff
15 08 03 U	ser		10							1	
	15 08			39E+01	14 82971					!	
	15 08			1 85E-01	D 197189		3			i	
97	15 09	18		4 60E-D2	0 05161	ı	i		· · · · · · · · · · · · · · · · · · ·	· ·	
98	15 09	54		2 67E-02	0 031401		1			†	

CAT, DeKalb, Illinois	3/12-15/1996				
Large fan	Outside tes	ting			
4202 Managerement Data	1904607/7902	4000 00 44 40 6	34 - 12-22		
- 1302 Measurement Data 1302 Settings	1804892/2803 •	1996-03-14 16 2	21 - Page 1	<u> </u>	<del></del> ;
1302 Senings	<del></del>	<del></del>	<del> </del>	<del></del>	
			_ <del></del>	<del></del>	<del></del>
Compensate for Water Vap	nteriorenee	NO	<del></del>	<del></del>	
Compensate for Cross Interfe		NO 0		<del></del> - <del>_</del> -	<u></u>
Sample Continuously	. YE		<del></del>	<del></del>	
Pre-set Monitoring Period	<u></u>		<del> </del>	<del></del>	
Tra-set Monitorning Ferror	<u> </u>	<del></del>	<del> </del>	<del></del>	
Measure ,	<del></del>		<del> </del>		
Gas A Formaldehyde	- N	0 +	<del>   </del>		
Gas B Carbon dioxide	NO.		<del></del>	<del></del>	
Gas C Carbon monoxide		NO		<del></del>	<del> </del>
Gas D TOC as Propane		NO '			<del></del>
Gas E Sulfur hexafluoride	YE	· · · · · · · · · · · · · · · · · · ·		<del></del> _	
Water Vapour	NO NO			<del></del>	
	—·———————————			······	
Sampling Tube Length	15	0 ft		<del></del> ;	
Air Pressure	759 0 mr	·			
Normalization Temperature		DOF		<del></del> -	
1	<del></del>		1	<del></del> -	<del></del>
General Information	<del></del>				<del></del>
			<del></del>		
Start Time	1996-03-14 15 1	4	<del></del>	<del></del>	
Stop Time	1996-03-14 16 1				
Results Not Averaged		·	<del></del>	<del></del> -	
Number of Event Marks	<del></del> - ·	3	<del></del>		<del></del>
Number of Recorded Sample	s .	100		ı	
			, ,		
Samples Measured From 19	6-03-14 15 15		<del></del>		· · · · · · · · · · · · · · · · · · ·
			1		
Samp Time	Response Calibi	stion			
No hh mm ss Event	PPM Corre		<del></del>		
			<u> </u>	·	
1 15 15 04	2 55E-02 0 03	0144 Wind blov	ving back to	front, 18	degrees
2 15 15 47		5694 Wind sper			
3 15 16 22		4494			
4 15 16 58		2822	<del> </del>		
5 15 17 33	7 1DE-02 0 07	7789			
15 18 08 User	1		<u>.                                    </u>		
6 15 18 08	789E+00 83	5086 Rt side or	ıly, distribu	tion	<u> </u>
7 15 18 45	9 93E+00 10 5	3801	<u> </u>		
B 15 19 22	9 04E+00 9.58	2318			
9 15 19 57	1 16E+01 12 3	3759	1		
10 15 20 43	<del></del>	2971			
		3632 Avg	13 42278	200 00	Ave Eff
11 15 21 19	1 51E+01 16 t			00 00 70	

## Outside3\_Lg\_Fan

13 15 22 29	1.575 - 01	46 70400	577	20 41%	445 600	Marie File
15 23 04 User	1 57E+01	10 /9120	CV	2041%	112 69%	Max EII
14 15 23 04		10 50100	Bash mala	s, distributi		
15 15 23 40	8 91E+00	0.442047	Both Side:	s, aismoun	011	
16 15 24 16	2 40E+01			<del>,</del>	<del></del>	
17 15 24 53				<u></u>	<del></del>	<u> </u>
18 15 25 29	2 40E+01			! <u>-</u> i		
19 15 26 04	2 68E+01			<u> </u>		
20 15 26 39	2 59E+01			<u> !</u>		
21 15 27 15	2 97E+01				20.000	
22 15 27 50	1 97E+01			25 93084		
<del></del>			Std Dev	3 991621		
L=	1 91E+01			15 39%		Max Eff
24 15 29 01	1 42E+00			, , ,		! 
25 15 29 41	6 80E-02			<u>.                                    </u>		
26 15 30 17		0 490602				
27 15 31 23	6 44E-02			<u>, -</u>		<u> </u>
28 15 31 58	4 41E-02			, 1		
29 15 32 34,	6 54E-02			•		
30 15 33 09		0 071716		<u>·                                    </u>		
31 15 33 45	, 2 24E-01					
32 15 34 20	2 98E-01				•	
33 15 34 55		0 099258		+		
15 35 31 User	3			1		
34 15 35 31	1 41E-01	0 151101	Wind blov	ving left to	right, 270 (	degrees
35 15 36 07	7 64E+00	8 083559	Wind spee	d at 3 to 4	mph	
36 15 36 44	1 08E+01	11 47449	Rt side on	ly, distribu	tion	
37 15 37 20	8 80E+00	9 325				
38 15 37 55	7 28E+00	7 699208		1		
39 15 38 31	7 65E+00	8 094344				
40 15 39 06	1 13E+01	12 0137	Avg	9 526898	63 94%	Ave Eff
41 15 39 41	1 16E+01	12 33759	Std Dev	2 099929		
42 15 40 17	6 80E+00	7 187188	CV	22 04%		Max Eff
15 41 11 User	4			· · · · · · · · · · · · · · · · · · ·		
43 15 41 11	1 91E+01	20 52392	Both side:	s, distributi	on	<del></del>
44 15 41 49	1 29E+01	13 74422				•
45 15 42 27	2 07E+01			· · · · · · · · · · · · · · · · · · ·		
46 15 43 05	2 11E+01	22 7367		<del>-</del>		<del></del>
47 15 43 40	1 56E+D1	16 68204				<del> </del>
48 15 44 18	1 69E+01	18 10455		1 1		 i
49 15 44 54	3 18E+01			21 35597	73 29%	Ave F#
50 15 45 32	2 07E+01			6 280786		
51 15 46 07;.	1 88E+01			29 41%		
52 15 46 42	7 84E-01				43/0	-718 <u>~ 511</u>
53 15 47 22 .	8 09E-02			<u>'</u>		<del></del> -
54 15 47 5B	2 89E-02			· 		
55, 15 48 33	3 18E-02					<u> </u>
56, 15 49 09	2 45E-02			· · · · ·		
57 15 49 44	2 80E-02			<u>.                                    </u>		·
58 15 50 31	3 83E-02					<u> </u>
				I		; 
59, 15 51 06	4 15E-01	0 43B191		· - ·	<del></del> -	·
15 51 41 ,User 1	5			·		<u>'</u>

## Outside3\_Lg\_Fan

		<del>-</del>				
60 15 51 41	3 86E+00	4 065401	Wind blov	ving right to	left, 90 de	egrees _
61 15 52 19	1 D8E+01	11 47449	Wind spec	ed at about t	to 7 mph	
62 15 52 55	7 39E+00			ly, distribut		·
63 15 53 30	1 20E+01	12.76986				
64 15 54 05	9 39E+00	9 957876		<del></del> i		
65 15 54 41	8 59E+00	9.099985	Avg	9 709183	65 16%	Ave Eff
66 15 55 16	9 23E+00	9 786148	51d Dev	1 980177	52 46%	Min Eff
67 15 55 52	6 68E+00	7.059286	CV	20 39%	85 70%	Max Eff
15 56 27 User 1	6,		<u> </u>	1		
68 15 56 27	8 92E+00	9 453638	Both side	s, distributio	on	
69 15 57 03	1 16E+01	12.33759		1		
70 15 57 38	1 12E+01	11 9058			i	
71 15 58 13	1 21E+01	12 878				
72 15 58 49	7 78E+00	8 233269		,		
73 15 59 24	1 43E+01	15 26476		, "!		
74 15 59 59	7 57E+00	8.008876	Avg	12 79517	43 91%	Ave Eff
75 16 01 06	1 21E+01	12 878	Std Dev	4 06718	27 48%	Min Eff
76 16 01 41	1 94E+D1	20 85502	CV	31 79%	71 57%	Max Eff
77 16 02 19	1 44E+01	15 3736	<del></del>			
78 16 02 57	6 02E-02	0 06648				
79 16 03 35		0 032552		<u> </u>		:
80 16 04 10		0 029202		<del></del>		i
81 16 D4 45	2 35E-02	0 02805		T		
82 16 05 21	5 93E-02	0 065537	_	<del></del>		ļ
16 05 56 User	7				· · · ·	ı
83 16 05 56	2 26E-01	0 24014	Wind blov	ving front to	back, 0 c	legrees_
84 16 06 32	1 11E+01			ed at about		
85 16 07 09	1 13E+01	12 0137	Rt side or	ily, distribut	ion	
86 16 07 45	1 14E+01	12 12164				
87 16 09 49	1 17E+01	12 44561				
88 16 10 24	1 19E+01	12 68175				1
89 16 11 DD	1 17E+01	12 44561	Avg	12 08314	81 09%	Ave Eff
90 16 11 35	9 81E+00			0 75209		Min Eff
91 16 12 10	1 20E+01	12 76986		6 22%		Max Eff
16 12 46 User	В			1		:
92 16 12 46	1 12E+01	11 9058	Both side	s, distributi	on	,
93 16 13 21	2 39E+01					
94 16 13 59	2 22E+01	23.95924		: 1		İ
95 16 14 34	2 18E+D1	23 51423		<del></del> ;		
96 16 15 09	2 21E+01	23 84794		<del></del> -		
97 16 15 45	2 00E+01	21 51808		1		1
98 16 16 20	1 61E+01	17-22854		22 19447	76 16%	Ave Eff
99 16 16 58	2 25E+01	24 29335		3 25377		Min Eff
100 16 17 36	1 62E+01	17.33793		14 66%		Max Eff
	, 0,2,2,181		<del></del>		1010	IMIGN FIL

CAT, DeKalb, Illinois	3/12-15/1996		<u> </u>
Calibration done in the lat	- <del> </del>		
1200 Monouroment Date	4.00400010000		
<ul> <li>1302 Measurement Data ——</li> <li>1302 Settings</li> </ul>	<u> 1804892/2803 - 1996-0</u>	3-08 10 31 - Page	<u>1- : </u>
1302 Settings		<u> </u>	<del>_</del> !
	= <del></del>	•	<del></del>
Compensate for Water Vap In	terference NC	<del></del>	
Compensate for Cross interfer			e B&K used in
Sample Continuously	YES		er-Greene,Caterpillar
Pre-set Monitoring Period	ND ND	March '96	
1	- 140	1	351703
Measure	:		<del> </del>
Gas A Formaldehyde	NO	<del></del>	<del></del>
Gas B Carbon dioxide	NO	<u> </u>	1 1
Gas C Carbon monoxide	NO	•	•
Gas D TOC as Propane	NO	· · · · · · · · · · · · · · · · · · ·	
Gas E Sulfur hexatioonde	YES		
Water Vapour	NO		
Sampling Tube Length	15 O ft		<u> </u>
Air Pressure	768 9 mmHg	į į	,
Normalization Temperature	74 5 F		ï
			1
General Information			1
		<u> </u>	· (
Start Time	1996-03-08 08 21		,
Stop Time	1996-03-08 10 29		
Results Not Averaged		····	
Number of Event Marks	10	<del></del>	
Number of Recorded Samples	209	<del></del>	<del></del>
Formulas Managed Frag 4000	00.00.00.04		
Samples Measured From 1996	-03-08 08 21	· -	<u> </u>
Samp Time	SF6 Gas	<del></del>	<del></del>
No hh mm ss Event	<del> </del>		<del></del>
THE OWN 25 EVENT	ppm		<del></del>
1 B 21 35	1 91E-02	<u>+</u>	<del>-i</del>
2 8 22 18,	1 33E-02	· · · · · · · · · · · · · · · · · · ·	
3 8 22 53	1 52E-02	<del></del>	
4 8 23 28	1 15E-02		
5 8 24 04	1 58E-02		<del></del>
6 B 24 39	1 85E-02	<u> </u>	<del></del>
7 B 25 14	1 55E-02 Room air, ve	nt lab	
8 8 25 50	1 37E-02 Average =	0 0154	<del></del>
9 B 26 25	1 63E-02 Std Dev €	0 0024	1 1
8 27 01 User 1	<del></del>	<del>-                                    </del>	· · · · · · · · · · · · · · · · · · ·
10 B 27 01	1 17E-02	E	<del>'</del> +
11 8 27 36	9 69E-03		
12 8 28 12	1 17E-02		<del></del>

13 8 28 47	1 18E-02 N2 supply had
14 8 29 22	
	9 41E-03 Average = 0 0116
	1.0 IE-02 010 DCV - 0 0020
31 04 User	2
16 8 31 04	1 83E-02 N2 supply bag 2
17 8 31 40	1.65E-02
32 15 User	3,
18 8 32 15	1 43E-D2
19 8 32 50	8 97E-03 N2 only in calibration bag
20 8 33 26	1 54E-02 Average = 0 0120
21 8 34 01	9 16E-03 Std Dev = 0 0034
34 37 User	4
22 8 34 37	1 QQE-02 4
23 8 35 12	1 62E-02
24 8 35 48	1 66E-02
25 B 36 23	1 20É-02
26 8 36 59	1,25E-02
27 8 37 34	1 44E-D2
28 8 38 10	1.38E-02
29 8 38 45	1 34E-02
30 8 39 20	1 50E-02
31 8 39 56	1 40E-02
32 8 40 51	1 24E-02
33 8 41 26	1 55E-02
34 8 42 01	1 12E-02
35 8 42 37	1 15E-02
36 8 43 12	1 34E-02
37 8 43 48	1 66E-02
38 8 44 23	1 34E-02
39 8 44 58	1 39E-02
40 8 45 34	1 30E-02
41 8 46 09	1 71E-02
42 8 46 45	1 62E-02
43 8 47 20	1 42E-02
44 8 47 55	1 61E-02
45 8 48 31	1 D3E-D2
45 8 49 06	1 20E-02
47 8 49 41	1 40E-02
48 8 50 28	1 45E-02
49 8 51 03	1.17E-02
50 8 51 38	1 44E-02 :
51 8 52 14	1 83E-02
52 8 52 49	1 60E-02
53 8 53 25	1 21E-02
54 8 54 DD.	1 19E-02
55 8 54 35	1 50E-02
56 8 55 11	1 43E-02
57 8 55 46	1 32E-02
56 8 56 22	8 05E-03
59 8 56 57	1 70E-02
60 8 57 32	1 33E-02
55 0 07 32	

611 3 08	* 77F 00
	1 77E-02
	1 45E-02
63 8 29 19	1.35E-02
64 8 per 54	1 45E-02
65 9 04 01	1 53E-02
66 9 01 36	1 47E-02
67 9 02 11	1 37E-02
68 9 02 47	1.29E-D2
69 9 03 22	1 26E-02
70 9 03 58	1 63E-02
71 9 04 33	1.21E-02
72 9 05 09	1 53E-02
73 9 05 44	1,54E-02
74 9 06 19	9 10E-03
75 9 06 55	1 33E-02
76 9 07 30	1 19E-02
77 9 08 06	1 53E-02
78 9 08 41	1 79E-02
79 9 09 16	1.24E-02
80 9 09 52	1 72E-02 Room air, vent lab
81 9 10 47	1 61E-02 Average = 0 0141
82 9 11 22	2 07E-02 Std Dev = 0 0021
	5.
83 9 11 57	1 89E-02
84 9 12 33	1.01E-02
85 91308	1 59E-02
86 9 13 43	1 18E-02
87 91419	8 29E-03 N2 in a calibration bag
88 9 14 54	1 12E-02 Average = 0 0113
89 9 15 30	1 06E-D2 Std Dev = 0 0025
90 9 16 05	1 66E-02
91 9 16 40	1 27E-02
92 9 17 16	1 37E-02
93 9 17 51	1 23E-02
94 9 18 27	1 31E-02
95 9 19 02	1 50E-02
96 9 19 37	1 63E-02
97 9 20 24,	1 36E-02
98 9 20 59	1 63E-02
99 9 21 34	1 54E-02
100, 9 22 10	1 51E-02
101 9 22 45	1 58E-02
102 9 23 20'.	1 50E-02
1031 9 23 561.	, 1 26E-02
104: 9 24 31	1 25E-02
105 9 25 07	4 005 00
	1 71E-02
106 9 25 42	
107 9 26 17	1 55E-02
108 9 26 53	1 69E-02
109 9 27 28	1 60E-02
110 9 28 04	1 37E-02

<del></del>								
111	9 28 39	t.	23E-02				1	
112	9 29 14		94E-D2				1	-
113	9 29 50		18E-02					
114	9 30 56		47E-02		<del></del> .		1	<del>'i</del>
115	9 31 32		1 44E-D2				<del></del>	1
116	9 32 07		52E-02				<del>!                                    </del>	<del>-!</del>
117	9 32 42		47E-D2				<del></del>	<del></del>
118	9 33 18,			1				<del>-</del>
			38E-02				<u> </u>	<del></del>
119	9 33 53		43E-02	<del></del>			•	<del></del>
120	9 34 29		2 10E-02				·	
121	9 35 04		2 41E-02	<del></del>		!	<u>i</u>	<u> </u>
122	9 35 40		2 09E-02			!	·	
123	9 36 15		22E-02			<u> </u>		1
124	9 36 50	1 4	82E-02	31	· -		<u> </u>	
125	9 37 26		74E-02	; · · · · · · · · · · · · · · · · · · ·		<u> </u>	1	
126	9 38 01		91E-02				<del> </del>	7 1
127	9 38 37		170E-02				[	<del>                                     </del>
128	9 39 12		50E-02			<del></del>	<del></del>	<del></del>
129	9 39 47		54E-02					
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9 41 18 L			- 01E-02	<del></del>			<del></del>	<del>-                                    </del>
			706-70		_	· <del>-</del>	1	+
131	9 41 18		85E-02				<u></u>	<u></u> .
132	9 41 53		90E+00			<u>!</u>	·	<del></del> -
133	9 42 31		90E+00		<del></del>		· <del></del>	
134	9 43 06			2 ppm SF6			·	<u>'</u>
135	9 43 42			Average =				<u> </u>
136	9 44 17	1	90E+00	Std Dev =	0 0052		T	,
137	9 44 53	1	_91E+D0			i	1	1
138	9 45 28		ELECT	20 ppm SF	5 in N2	<u> </u>		i
139	9 46 D3		67E+01	Average =	18 6567			1
140	9 46 39		A CON	Std Dev =	0 0577		<del></del>	1
141	9 47 16 .		01E-02					
142	9 47 54		93E-02		_ <del></del> _	<del></del>		<del></del>
143	9 48 30		52E-02			<del></del>		<del> </del>
144	9 49 05					! ? * · ·	·· •	
			1 76E-02					<u> </u>
145	9 49 40	<del></del> _	1 96E-02				_'	
146	9 50 27		1 87E-D2					<del></del>
147.	9 51 02		1 29E-02			<del> </del>	<u> </u>	
148	9 51 37		1 73E-D2			<u> </u>	<u> </u>	1
149	9 52 13 .		1 29E-02			<u> </u>		<u> </u>
150	9 52 48,		4 00E-02				<u> </u>	
151	9 53 23		2.28E-02				!	
152	9 53 59		1 40E-02	<del></del>		<del></del>	1	1
153	9 54 34,		176E-D2			<del></del>		
9 55 10 L		7					<del></del>	
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156	9 56 25		34E+01			<del></del>	<del></del>	<del>_</del>
					6 up N/3	<del></del>	<del>+</del>	<del></del>
157	9 57 01:			25 ppm SF		<del> </del>	<del>'</del>	<del></del>
158	9 57 36			Average =			<del></del>	
159	9 58 12	<u></u>	34E+01	Std Dev≃	0 0632	<del></del>	<u> </u>	!

150 9 58 47	A 355				<del></del>
	9 25E				
161 9 59 27 10 00 03 User	281E-		<del></del>	<del></del>	
	8			<del></del>	
162 10 00 03	3.28E40.	3 ppm Si			<u></u>
163 10 01 14	5 32E C:	rerage =	53 D333		<u> </u>
164 10 01 50	5.31Etg)	d Dev =	0 2082		·
165 10 02 25	1 2 20E-01				
166 10 03 05	5 32E-02				
167 10 03 41	, 3 47E-02		<b>!</b>		
168 10 04 16	2 47E-02				
10 04 52 User .	9				
169 10 04 52	2 17E-02			1	
170 10 05 27 .	2 13E-02	7			1
171 10 06 03	2 24E-02 N	2 supply b	ag		1
172 10 06 38	1 85E-02 A		0 0210		
173 10 07 13	2 12E-02 S	old Dev =	0 0015		
10 07 49 User	10	<del></del>		1	
174 10 07 49	2 52E-02	1	1	<del></del>	
175, 10 08 24	A. LOE VOL S	9 7 ppm S	F6 in N2		
176 10 09 05	Part 1017	verage =	77 <del>9</del> 333		
177 10 09 40	7 SIE TOU	Std Dev =	0 2082	<del></del>	
178 10 10 35	3 15E-01			<del>- ,                                   </del>	· · · · · · · · · · · · · · · · · · ·
179 10 11 15	6 65E-02			1	<u> </u>
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185 10 14 48	2 27E-02	·	<del></del>	<del>-                                    </del>	
186 10 15 23	1 95E-02			i	
187 10 15 5B	2 20E-02	·	;	· · ·	····
188 10 16 34	2 53E-02			<u> </u>	<del></del>
189 10 17 09	1 88E-02			<del></del>	- <u>.                                    </u>
190 10 17 45	2 24E-02				<del> </del>
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192 10 18 56	1 74E-02				· · ·
193 10 19 31	1 88E-02		<del></del> -		<del></del>
194 10 20 17	1 17E-02				1
195 10 20 53	1 63E-02				
196 10 21 28	1 72E-02	<del></del>	<del></del>	<del></del>	<del></del>
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199 10 23 14.	1 65E-02	<del></del> 1		<del></del>	<del></del>
200 10 23 50	2 18E-02	<del></del> :	<del>+</del>	<del></del>	<del></del>
201 10 24 25	1 69E-02	<u></u>		<del></del>	
202 10 25 01	1.69E-02	<u> </u>			
4	1.09E-02		<del></del>	<u></u>	
203 10 25 36		<del></del>	<del></del>	<del>  </del>	<del></del>
204 10 26 12	1 47E-02			—— <del>——</del>	<u></u>
205 10 26 47	1 99E-02	· · · · · · · · · · · · · · · · · · ·	<del></del>		
205 10 27 22	1 71E-02	2000 010	eni lah		<del> </del>
207 10 27 58	2 005-021	Room air, v	CIN IND	{	<u> </u>

Cali 1% 7

208	10 28 3		1 83E-02 Avera			
209	10 29 (	9	1 B2E-02 Std [	Dev = 0 0031		<del></del>
						'   - <del>  </del>
			1	<u> </u>		<u> </u>
			ta for ECTB#1267			<u> </u>
	Concent	ra Respons		Correction		<u> </u>
	<u>.                                    </u>	0 0012		0 015964		
;		2 1 903		2 001218		
		20 18 666		20 04617		
	:	25 23 400		25 29743		
	50	3; 53 033	33'	59 99566		
	99	7, 77 933		•		1
				4!		· · · · · ·
				1		
	<b>8</b> 0 —	. —	Response	•,		
	70			1	1	
	,, L.—					
E	1	1				
E	50	<del></del>		1		
нзе рукп	50 -					
пуда врем	1					
Response ppm	50 -					
SK Response ppm	50 40 30					
B&K Response ppm	50 40 30 20					
B&K Response ppm	50 40 30 20		•			
B&K Response ppm	50 40 30 20 10	20			BO BO	
B&K Response ppm	50 40 30 20	20	40	60	в0	100
B&K Response ppm	50 40 30 20 10	20		60	B0	